Coupled Lateral-Torsional Vibration Analysis of a Geared Shaft Rotor System using a Complex Rotor Variable Approach

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Though there have been numerous studies on both rotor dynamics and gear dynamics, the studies on geared rotor dynamics have been rather recent. The study of the dynamic behavior of geared rotor systems usually requires that torsional and lateral vibration modes be coupled in the model, a problem not present for studies of rotors without gears. For rotors studied by many researchers, including Wang and Kirkhope,15 Dutt and Bakra,16 and Genta and Tonoli.17 The gyroscopic effect is what couples the forward and backward modes of rotors, as discussed by Kessler and Kim,18 For geared shaft systems, however, the torsional and lateral vibrations are generally coupled because the forces acting on the gears and the motion of the elements in one direction have influence in other directions. It has been demonstrated by Neriya et al.19 and Iida et al.20 that the predicted dynamic behavior of a geared shaft system, taking into account this cross coupling, could be different from that obtained using a conventional model of the system. Schwibinger and Nordamann21 studied the coupling effect between torsional and flexural vibrations in a geared rotor system that can affect the stability of the system and influence, for example, the stability threshold and damping constants in oil film bearings. It has been shown that the classical eigenvalue analysis, which ignores the coupling of torsional and flexural vibrations in gears, may lead to serious errors in the prediction of the stability onset speed, the critical speeds, and the natural modes.

1. INTRODUCTION

The study of dynamic behavior of geared rotor systems usually requires that torsional and transverse vibration modes be coupled in the model, a problem not present for studies of rotors without gears. Early geared rotor dynamic models concentrated on the effects of mass imbalance and eccentricity of the gear on the shaft, virtually neglecting the actual dynamics of gear mesh. Mitchell and Mellen included lateral vibration of the shafts and their bearings in their models.

In their study, they considered the coupled vibration of shaft-bearing systems through experimental models.2 included influence coefficients at each gear mesh by using the Holzer method for torsional vibrations and the Myklestad-Prohl method for lateral vibrations, thus obtaining critical speeds and a forced vibration response.

The backlash detection and its influence in geared systems have been studied by Sarkar et al. Ambili and Fregolent determined modal parameters of spur gear systems using the harmonic balance method. A study of designing compact spur gears, taking into consideration tooth stress and dynamic response, was conducted by Lin et al. The dynamic behavior of spur gears to varying mesh stiffness and tooth error has been studied by Kuan and Lin. A mathematical model for gear geometry errors and mounting errors has been developed by Velex and Maatar.7 Theodossiades and Nastiavas8 studied the nonlinear behavior of gear systems with backlash and varying stiffness. Theodossiades and Nastiavas also studied the nonlinear influence of bearings characteristics on gear pair systems.9 A modal analysis of compliant multibody gear systems has been carried out by Vinayak and Singh.10 Shen et al.11 performed coupled torsional-lateral vibration of the unbalanced rotor system with external excitations through numerical simulation.

Rotating shafts tend to bow out and whirl at certain speeds called the critical speed or whirling speed. Various factors that can cause the shaft to whirl are mass unbalance of the rotating system, gyroscopic forces, unsymmetrical stiffness, etc.12–14 They have conducted studies on the dynamics of rotors and discussed related assumptions. Modal analysis of undamped rotor systems and the gyroscopic effect of rotors has been studied by many researchers, including Wang and Kirkhope,15 Dutt and Bakra,16 and Genta and Tonoli.17 The gyroscopic effect is what couples the forward and backward modes of rotors, as discussed by Kessler and Kim. For geared shaft systems, however, the torsional and lateral vibrations are generally coupled because the forces acting on the gears and the motion of the elements in one direction have influence in other directions. It has been demonstrated by Neriya et al. and Iida et al. that the predicted dynamic behavior of a geared shaft system, taking into account this cross coupling, could be different from that obtained using a conventional model of the system. Schwibinger and Nordamann studied the coupling effect between torsional and flexural vibrations in a geared rotor system that can affect the stability of the system and influence, for example, the stability threshold and damping constants in oil film bearings. It has been shown that the classical eigenvalue analysis, which ignores the coupling of torsional and flexural vibrations in gears, may lead to serious errors in the prediction of the stability onset speed, the critical speeds, and the natural modes.