

Study on Linear Vibration Model of Shield TBM Cutterhead Driving System

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In this paper, a general linear time-varying multiple-axis (LTVMA) vibration model of shield tunnel boring machine (TBM) cutterhead driving system is established. The corresponding multiple inputs and multiple outputs (MIMO) state-space model is also presented. The linear vibration model is analysed, and the vibration-torque transfer function matrix and the vibration-torque static gain matrix are obtained. The linear vibration model is simulated, and the physical parameters' effects on the vibration response are investigated. A preliminary approach is proposed to reduce vibration by increasing motor rotor inertia and viscous damped. The LTVMA vibration model provides a solid foundation for fault detection and diagnosis (FDD), as long as the health monitoring of cutterhead driving system.

NOMENCLATURE

n	The number of cutterhead driving motors;	F_i	Elastic mesh force of the pinion- i and large gear
q	the reducer speed reduction ratio	r_i	radius of pinion- i
$J_{d,i}$	i -th induction motor rotor inertia after equivalent coupling- i	$M_{c,i}$	Elastic mesh torque of the pinion- i and large gear
$b_{d,i}$	i -th induction motor rotor viscous damped after equivalent coupling- i	p_i	Relative position of the pinion- i and large gear
$J_{z,i}$	Inertia of the i -th coupling between motor- i and reducer- i	k_i	Mesh stiffness of the pinion- i and large gear
$b_{z,i}$	Viscous damped of the i -th coupling between motor- i and reducer- i	c_i	Mesh damped of the pinion- i and large gear
$J_{r,i}$	i -th induction motor rotor inertia	$m_{p,i}$	Mass of the pinion- i
$b_{r,i}$	i -th induction motor rotor viscous damped	$k_{y,i}$	support stiffness of pinion- i
J_m	Large gear inertia	$c_{y,i}$	support damped of pinion- i
b_m	large gear viscous damped	m_g	Mass of the large gear
r_m	large gear radius	$k_{y,m}$	support stiffness of large gear
$\theta_{p,i}$	Angular displacement of i -th active pinion	$c_{y,m}$	support damped of large gear
θ_i	angular displacement of i -th motor rotor	y_i	The contact direction (line-of-action direction) vibration displacement of i -th pinion
θ_m	Large gear angular displacement	y_m^i	The large gear's contact vibration displacement along i -th pinion's line-of-action direction
ω_m	large gear angular speed	T_L	Shield TBM cutterhead's load torque
$J_{c,i}$	i -th pinion inertia after equivalent coupling- i	$k_{f,i}, c_{f,i}$	elastic mesh force coefficient of pinion- i
$b_{c,i}$	i -th pinion viscous damped after equivalent coupling- i	$k_{t,i}, c_{t,i}$	elastic mesh torque coefficient of pinion- i
$J_{w,i}$	Inertia of the i -th coupling between reduce- i and pinion- i	$i_{m,i}$	gear transmission ratio
$b_{w,i}$	Viscous damped of the i -th coupling between reducer- i and pinion- i	T_1	The resistant torque of soil and rocks
$J_{p,i}$	i -th pinion inertia;	T_2	The friction torque of soil and rocks which chafe with the front of the cutterhead
$b_{p,i}$	i -th pinion viscous damped	T_3	The friction torque of soil and rocks which chafe with the back of the cutterhead
$T_{e,i}$	i -th induction motor electrical magnetic torque	T_4	The friction torque of soil and rocks which chafe with bulkhead
$\omega_{p,i}$	Angular speed of pinion- i	T_5	The stirring torque of soil and rocks by cutterhead stirring rod
ω_i	angular speed of induction motor- i	T_6	The friction torque of cutterhead bearings and sealed chamber