

Dynamic Behaviour Analysis of a Dual-Rotor System Using the Transfer Matrix Method

B.B. Maharathi

Department of Mechanical Engineering, IGIT, Sarang - 759146, Orisa, India

P.R. Dash

Department of Mechanical Engineering, UCE, Burla, Orissa, India

A.K. Behera[†]

Department of Mechanical Engineering, NIT, Rourkela, Orissa, India

(Received 28 February 2001; accepted 16 April 2004)

This paper presents a general formulation for the problem of the steady-state unbalance response of a dual rotor system with a flexible intershaft bearing using an 'extended' transfer matrix method, where the transfer matrix assumes a dimension of (33×33). The validity of the formulation is established by comparing the results obtained through a computer program with closed form solutions available for some simple cases. Some interesting phenomena of steady-state whirl orbits of the dual rotor system are described.

[†] Member of the International Institute of Acoustics and Vibration (IIAV)

Nomenclature

$[A], [B]$ } Represent transfer matrices,
 $[C], [D]$ } each of dimension (33 × 33)

$[\bar{B}]$ – Bearing matrix
 C_{yy}, C_{zz} – Direct damping coefficients
 C_{yz}, C_{zy} – Cross coupled damping coefficients
 C_H, C_L – High and low rotor damping, respectively
 E – Modulus of elasticity
 e – Eccentricity
 $[F]$ – Field matrix
 $[\bar{F}]$ – Overall field matrix
 h – Thickness of the disk
 $[I]$ – Unit matrix
 I_T – Transverse mass moment of inertia of the disk
 I_P – Polar mass moment of inertia of the disk
 K_{yy}, K_{zz} – Direct stiffnesses
 K_{yz}, K_{zy} – Cross coupled stiffnesses
 K_H, K_L – High and low rotor stiffness, respectively
 l – Sectional length
 M_y, M_z – Moments about y - and z -axes, respectively
 M_H, M_L – High and low rotor mass, respectively
 m – Mass of the element per unit length
 m_b – Mass of the shaft at bearing station
 m_e – Unbalance mass
 $[P]$ – Point matrix
 $[\bar{P}]$ – Overall point matrix
 U_y, U_z – Unbalance components
 V_y, V_z – Shear force in y - and z -directions, respectively
 v, w – Deflection along y - and z -axes, respectively
 X, Y, Z – Fixed co-ordinate system
 $\bar{X}, \bar{Y}, \bar{Z}$ – Rotating co-ordinate system

$\{\bar{S}\}$ – Modified State vector at any station
 $\left. \begin{matrix} \{S\}_{OA}, \{S\}_{OB} \\ \{S\}_{OC}, \{Z\}_{OD} \end{matrix} \right\}$ – Represent end conditions
of the dual rotor system
 ω_m, ω_n – Inner and outer rotor speeds, respectively
 θ, ϕ – Slope in x - z and y - z planes, respectively
 β – Angular position of unbalance
 λ – Eigenvalue

Subscripts

m, n – Pertaining to the effect of inner and outer rotor speeds, respectively
 c – Cosine component
 cm, cn – Cosine components of inner and outer rotor, respectively
 s – Sine component
 sm, sn – Sine components of inner and outer rotor, respectively
 y – Along y -axis
 z – Along z -axis
 j – Pertaining to a junction

Superscripts

R – Right to a section
 L – Left to a section
 t – Transpose of a matrix

1. INTRODUCTION

With the ever increasing demand of large power and smaller gas turbine engines for aircraft propulsion, a two spool system with intershaft bearings is becoming a standard layout to accommodate the compressor and turbine rotors.